

# Modal Interaction and Mode Switching in Post-buckling Analysis of Plates Using a Multi-mode Finite Element Based Reduced Order Model

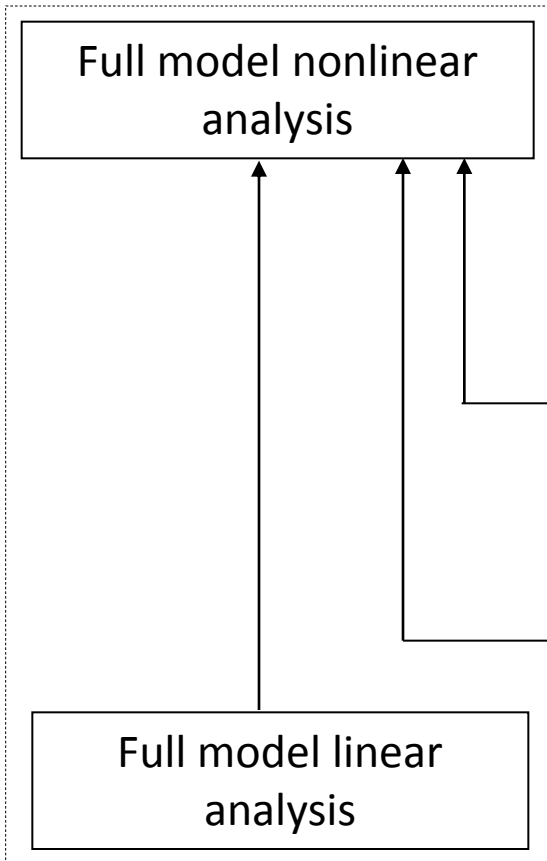
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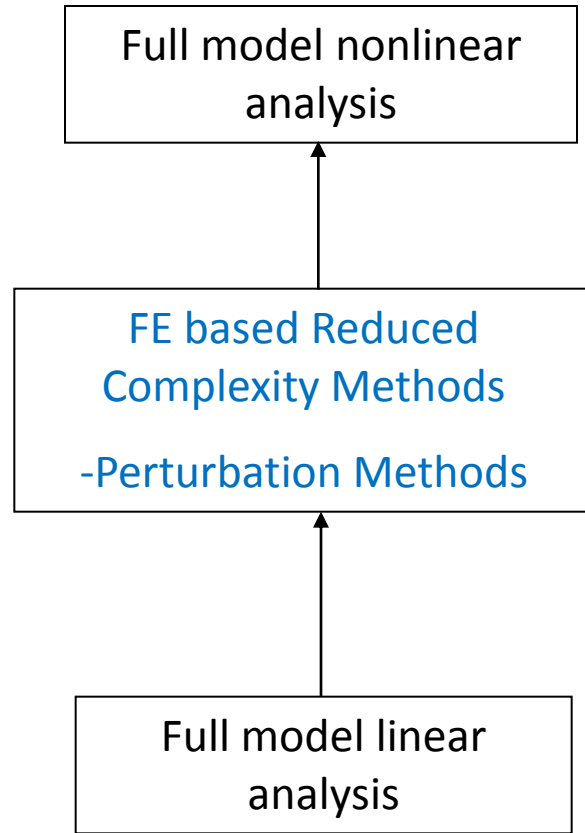
- Motivation
- Methodology
- Tested scenario
- Effect of imperfections on post buckling behavior
- Concluding remarks

- Trend towards optimization
- Imperfections dominate post buckling behavior
- Probabilistic analyses by traditional approaches can be costly
- Design tools accounting for probabilistic imperfection patterns require fast computation of multiple scenarios: use of reduced order models
  
- Reduced order modeling approach based on perturbation analysis introduced earlier shows good agreement with other more intensive analyses for static analysis
  
- Objective: Further definition of interesting test cases for nonlinear analysis to be used in the current extensions of the reduced order approach
  - Towards dynamic analysis
  - Towards probabilistic analysis

# Reduced order modelling: Motivation



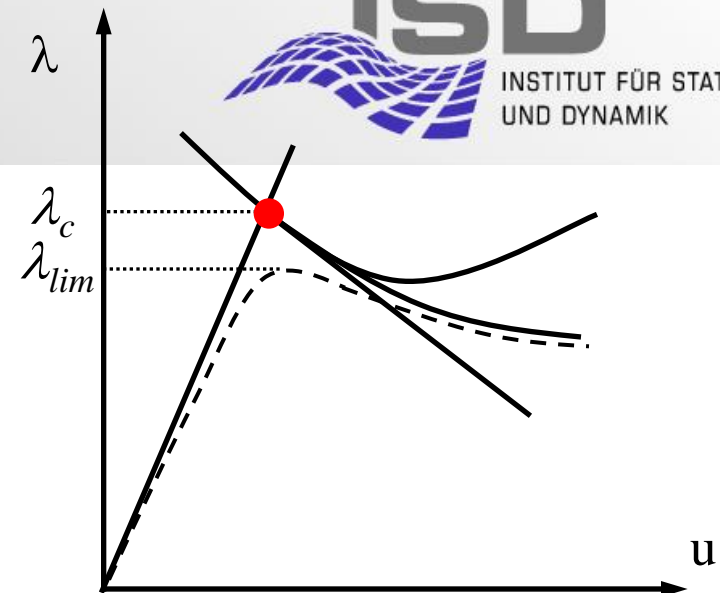
Traditional approach



Utilized approach

# ROM within FE framework basics: Perturbation procedure Stability

- Given a proportional load distribution
- we find the linear(ized) prebuckling path  $\mathbf{u} = \lambda \mathbf{u}_0$
- the buckling load and mode  $\mathbf{u}_1, \lambda_{cr}$
- The postcritical path is expanded about the critical point



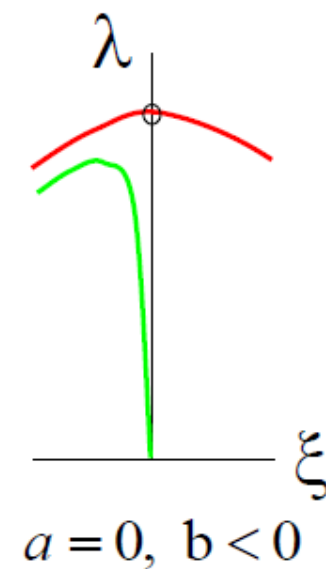
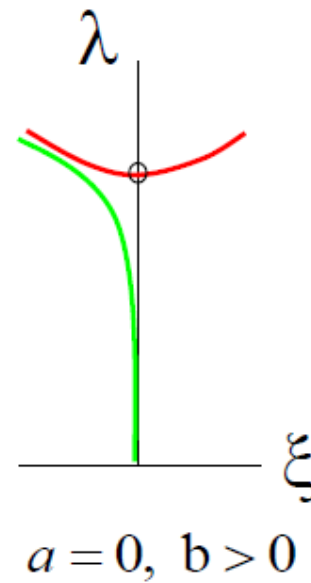
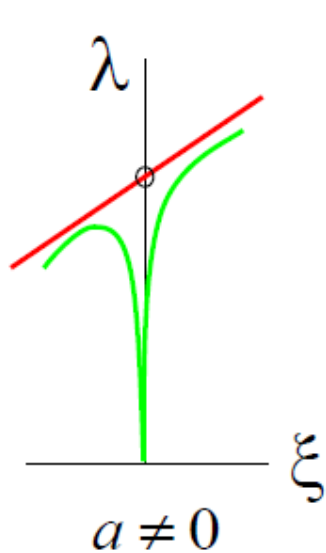
$$\mathbf{u} = \lambda \mathbf{u}_0 + \xi \mathbf{u}_1 + \xi^2 \mathbf{u}_2 + \dots$$

- We find the postcritical slope  $a = \frac{3\sigma_1 \cdot L_2(\mathbf{u}_1)}{2\sigma_1 \cdot \varepsilon_1}$
- and the postcritical curvature  $b = \frac{2\sigma_1 \cdot L_{11}(\mathbf{u}_1, \mathbf{u}_2) + \sigma_2 \cdot L_2(\mathbf{u}_1)}{\sigma_1 \varepsilon_1}$

- Given an imperfection pattern  $\xi$  the load-deflection path is found:  $\frac{\lambda}{\lambda_c} = \frac{1 + a\xi + b\xi^2}{1 + \frac{\xi}{\xi}}$

- Perturbation based reduced order model

$$\lambda = \lambda_c + a\lambda_c\xi + b\lambda_c\xi^2 + \dots$$



# FE Implementation: Reduction method for nonlinear static stability analysis

Fundamental path:  $\mathbf{K}_0 \mathbf{q}_0 = \mathbf{F}$

Buckling modes:  $[\mathbf{K}_0 - \lambda_i \mathbf{K}_G] \mathbf{q}_i = \mathbf{0}$

Second order modes:  $[\mathbf{K}_0 - \lambda_i \mathbf{K}_G] \mathbf{q}_{ij} = \mathbf{g}(\mathbf{q}_i, \mathbf{q}_j)$

Post-buckling coefficients:  $a, b$

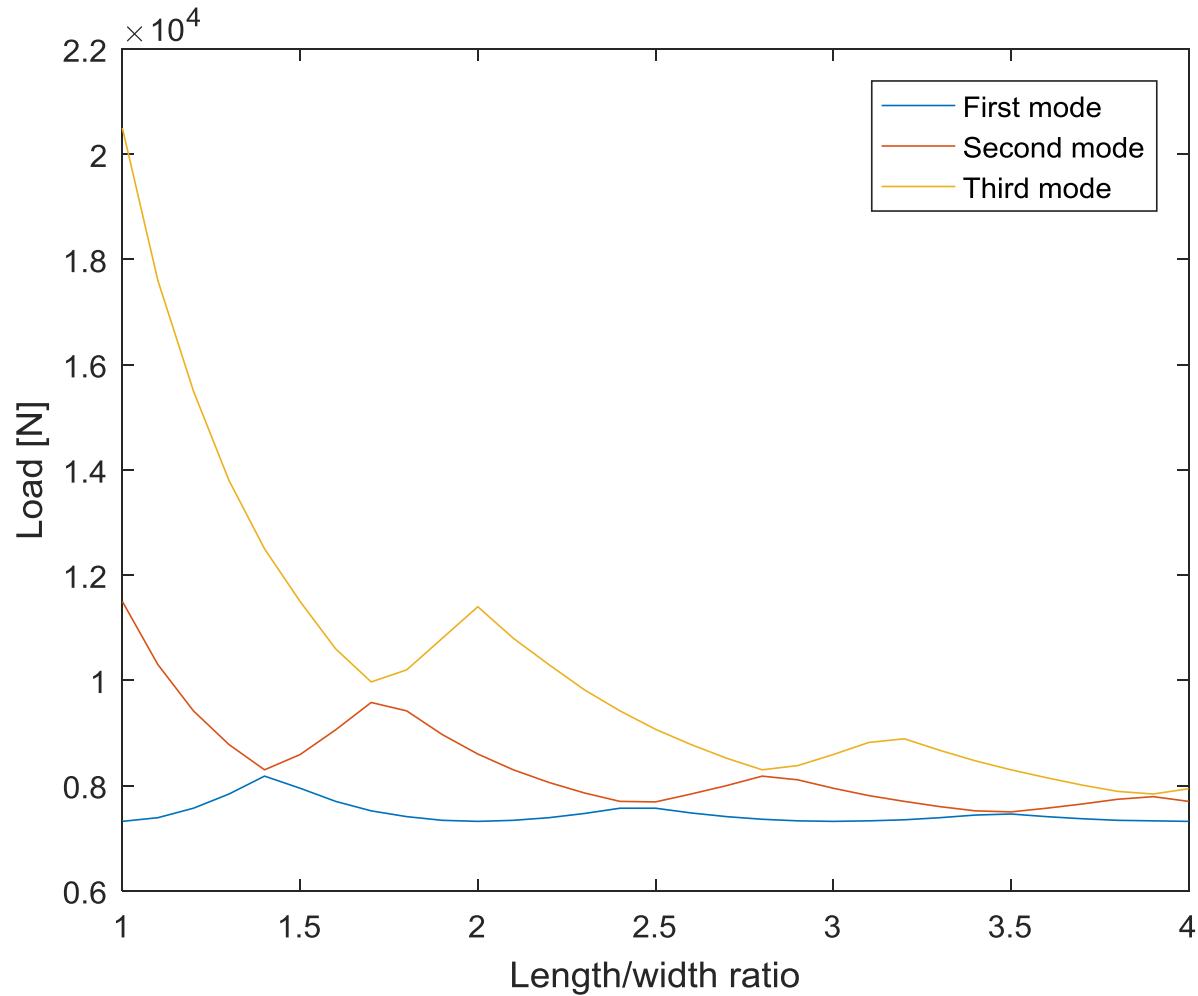
The reduced equation:  $(\lambda_c - \lambda)\xi + a_s \lambda_c \xi^2 + b_s \lambda_c \xi^3 = \dots$

Displacement field:  $\mathbf{q} = \mathbf{q}_0(\lambda) + \mathbf{q}_1 \xi + \mathbf{q}_2 \xi^2$

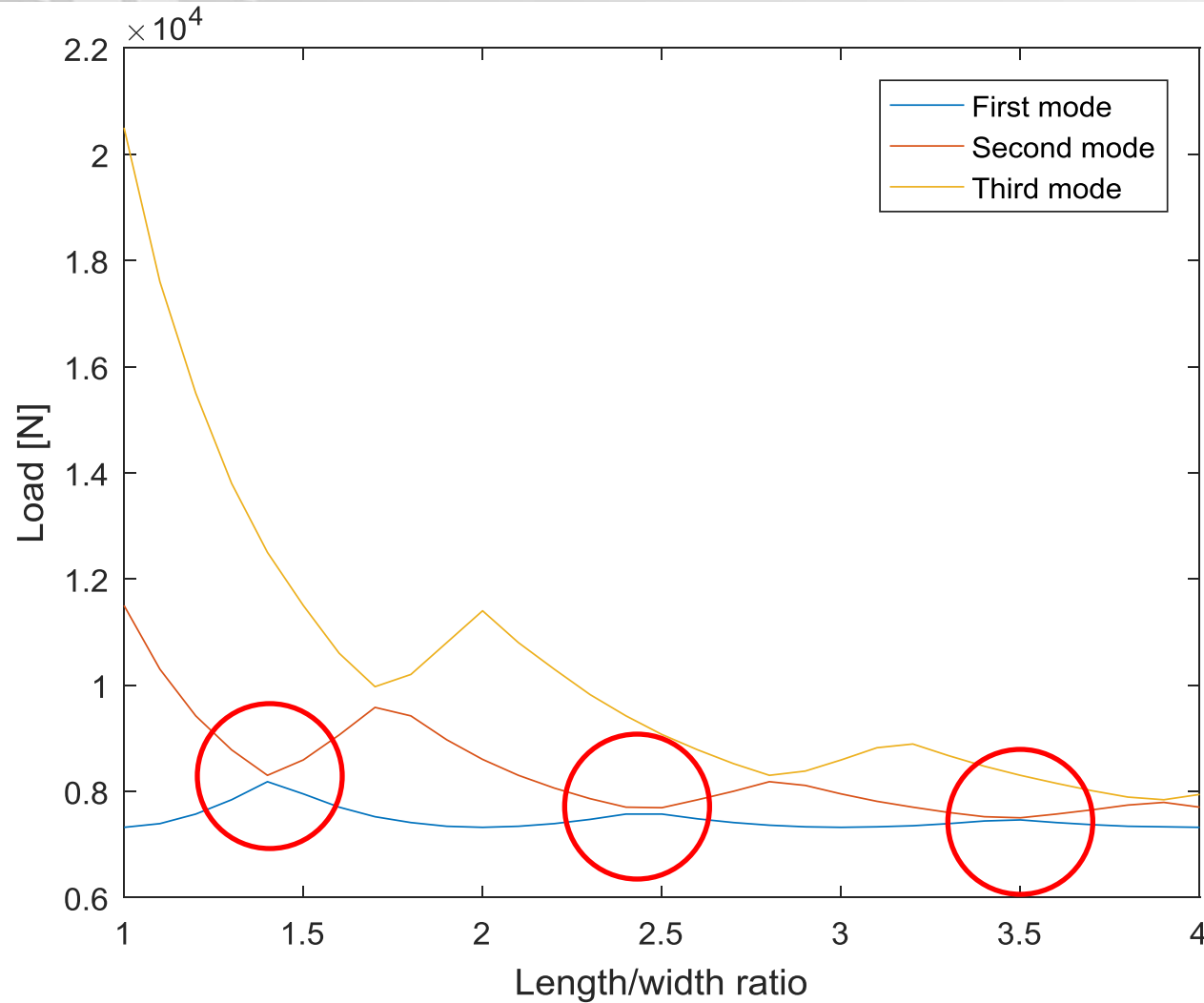
- Scenario to analyze modal interactions of simple plate configurations, making use of current Matlab implementation for isotropic structures:
  - Isotropic plates
  - Quasi-isotropic plates
- Plate configuration:
  - 1.5 mm, 12 layer “fully isotropic” laminate (Grédiac, 1998)
- Investigation of nonlinear behavior for
  - Varying height/width ratios
  - Varying imperfection amplitudes



# Mode interaction case

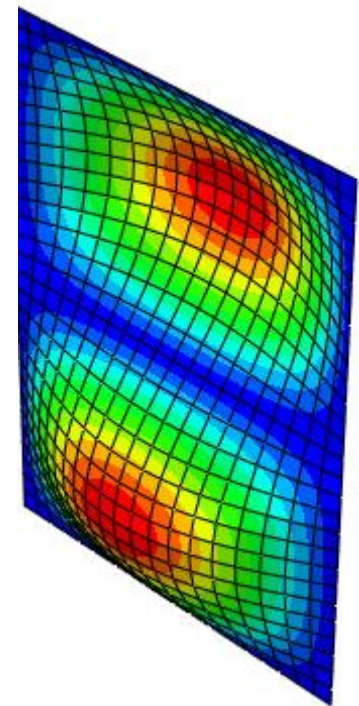
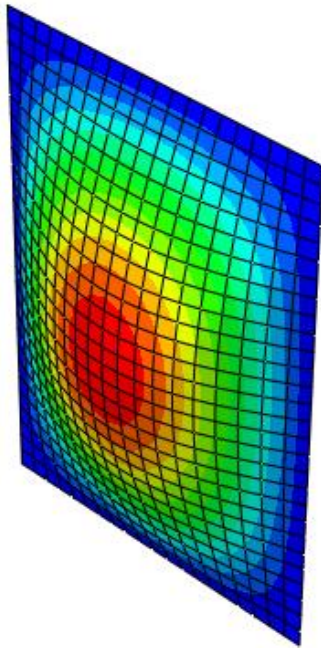


# Mode interaction case



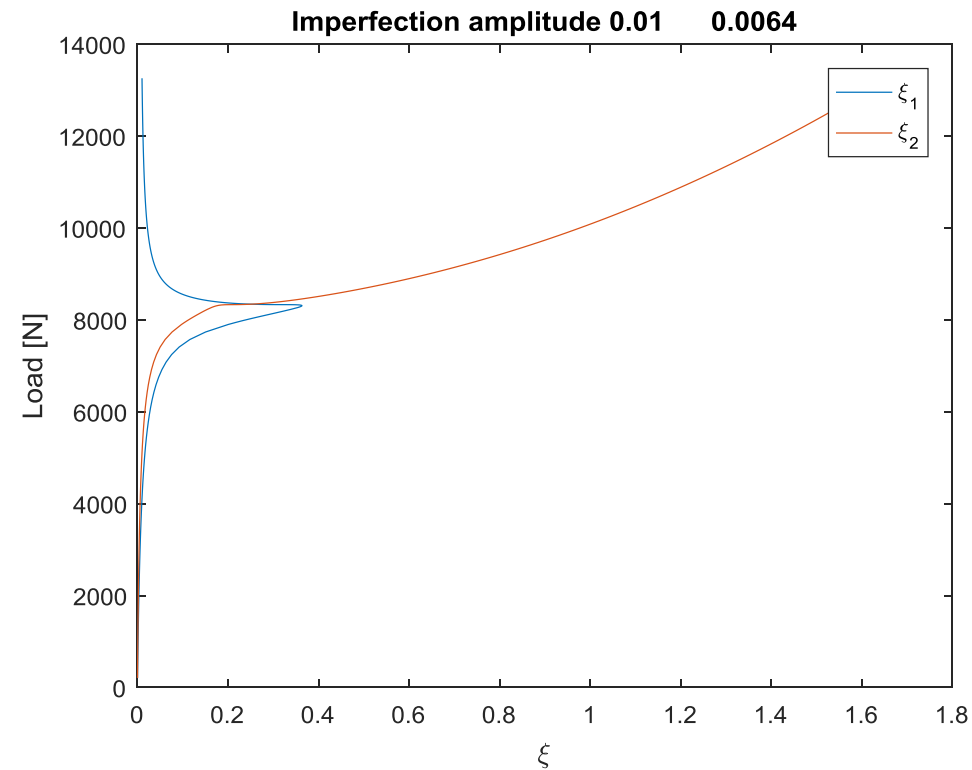
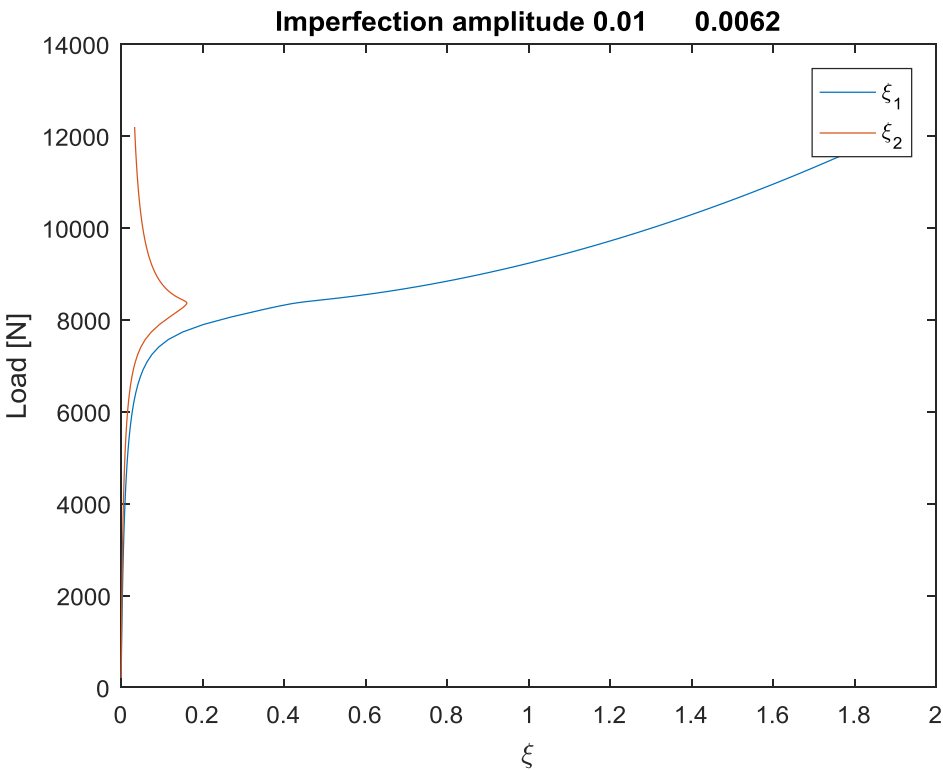
## Ratio of 1.4

- 1<sup>st</sup> and second mode are very close to each other
- Which mode gets triggered depends on the imperfections found in the structure

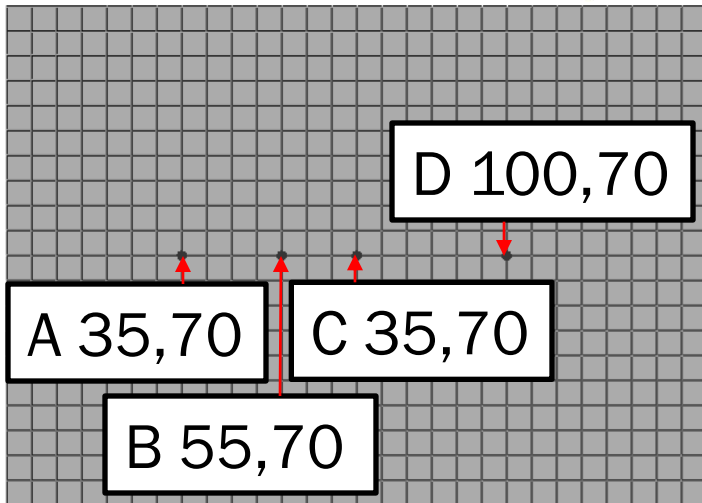
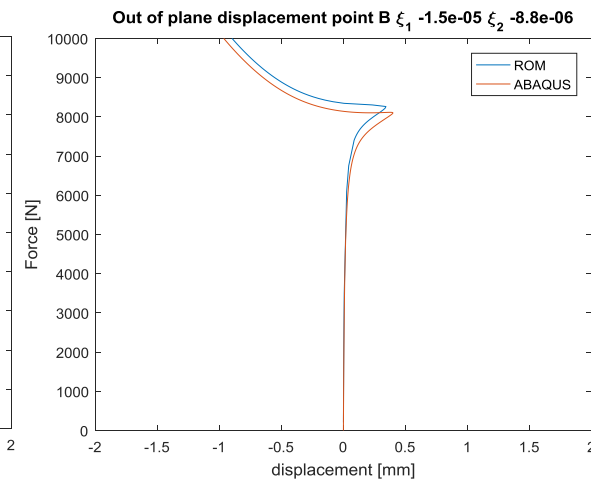
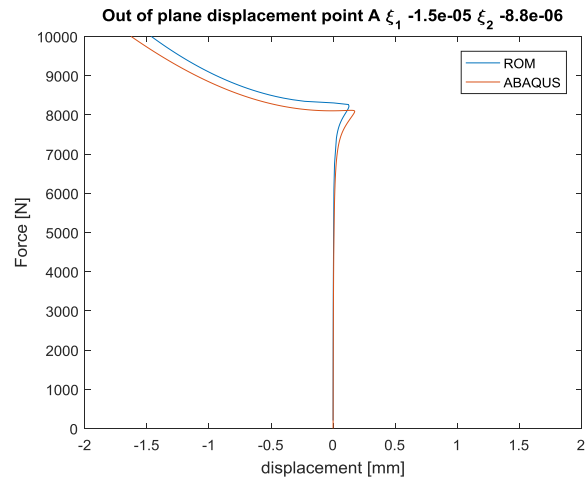
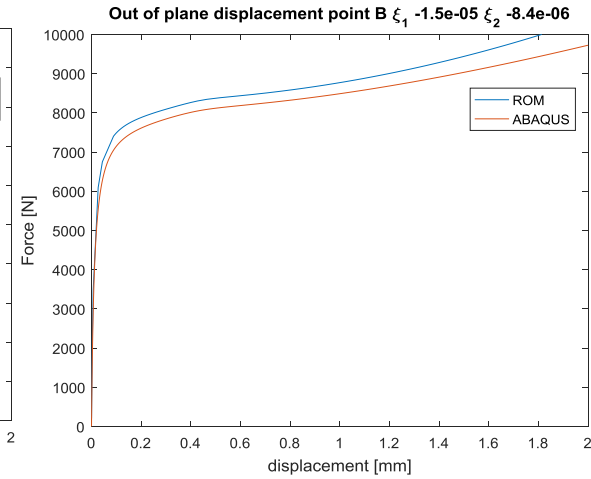
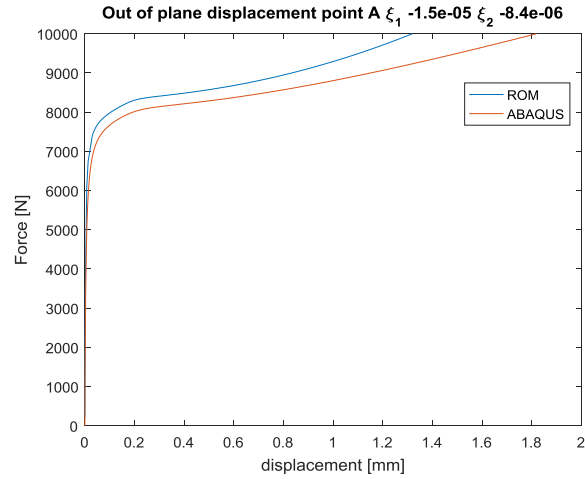
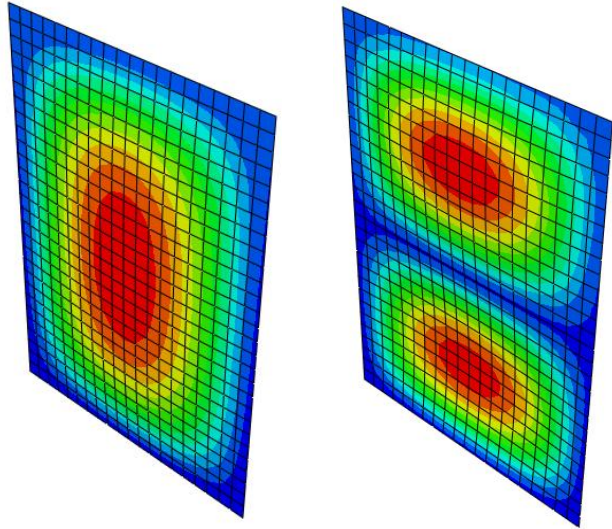


# Effect of imperfections on post buckling

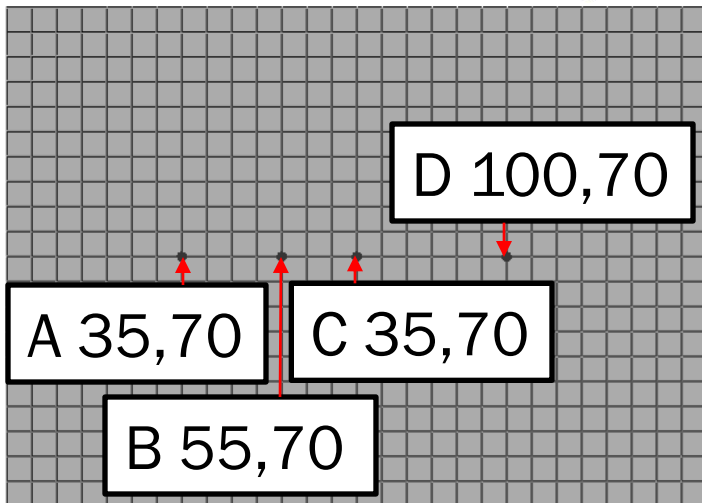
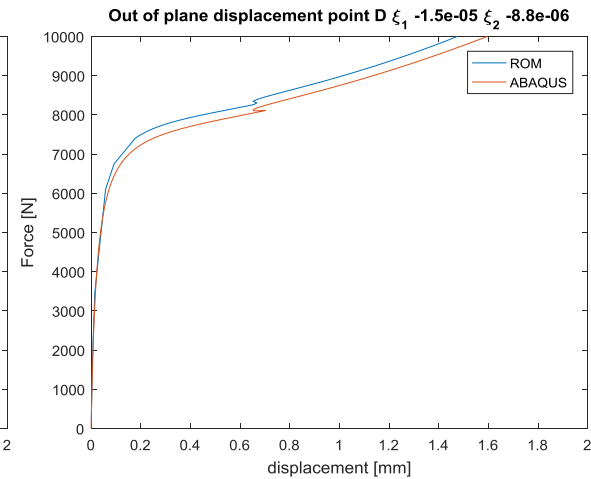
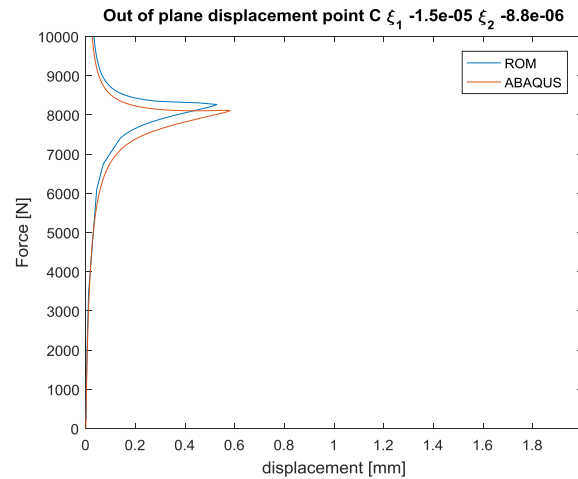
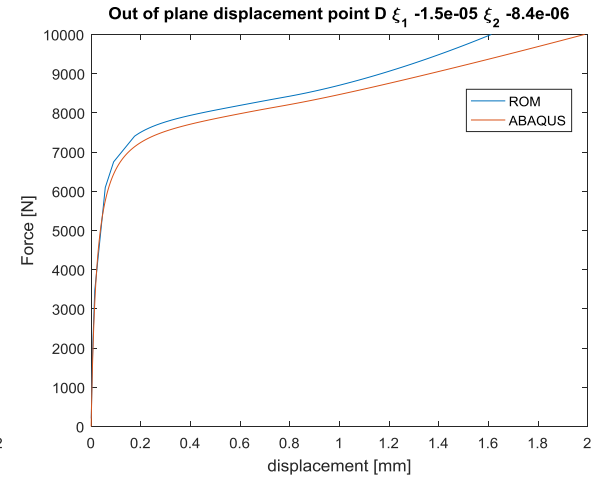
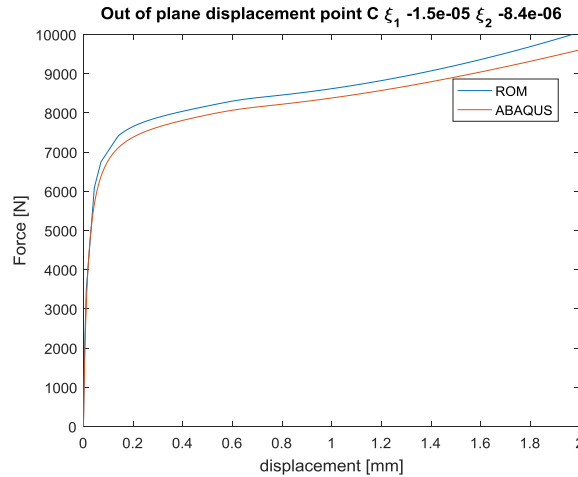
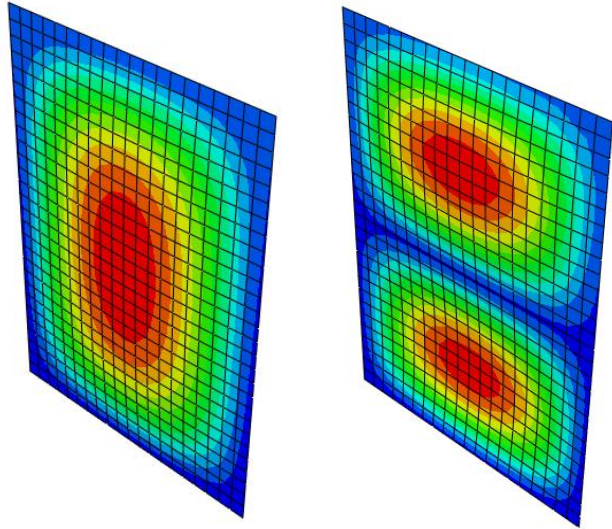
## Ratio 1.4



# Comparison with ABAQUS, points A & B

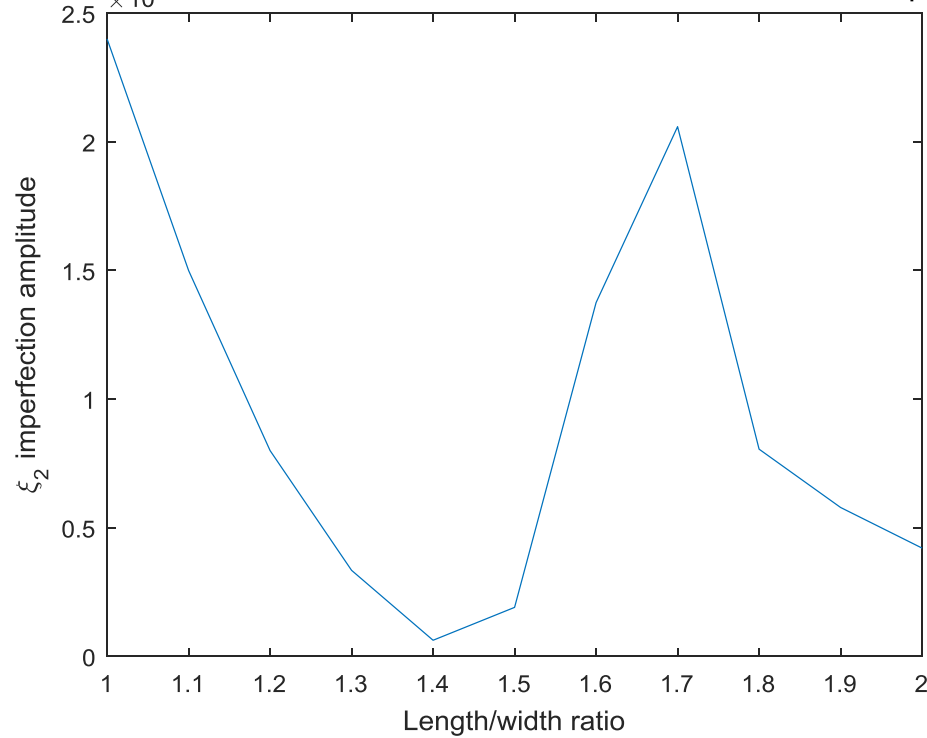


# Comparison with ABAQUS, points C & D

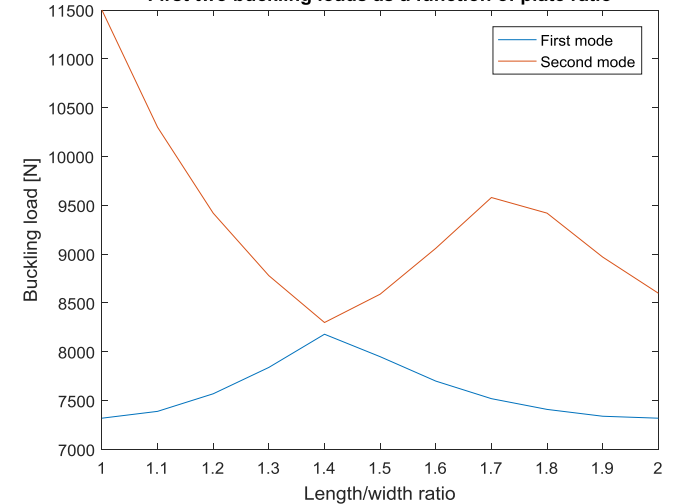


# Imperfection amplitude needed for mode switching

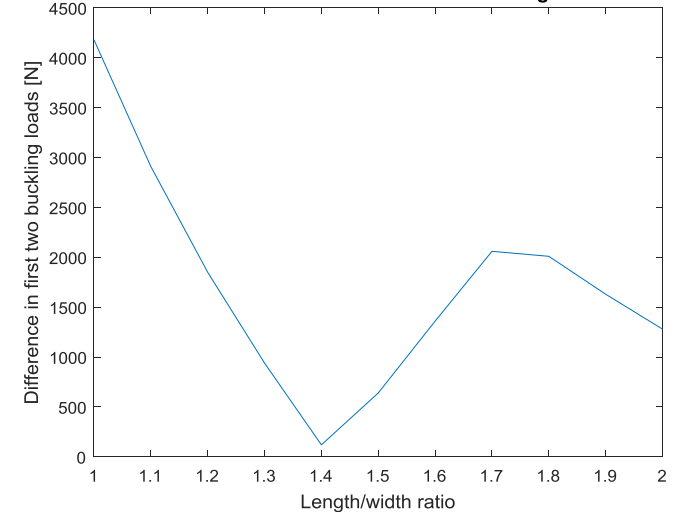
Imperfection amplitude needed to trigger second mode, constant  $\xi_1 = 1.5E-6$



First two buckling loads as a function of plate ratio



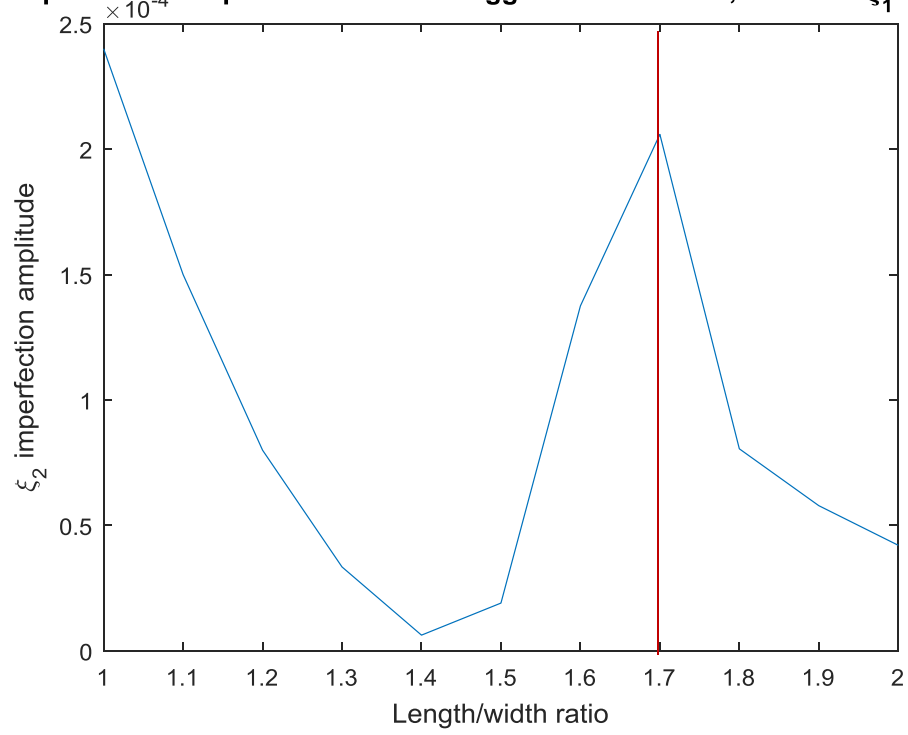
Load difference between first two buckling loads



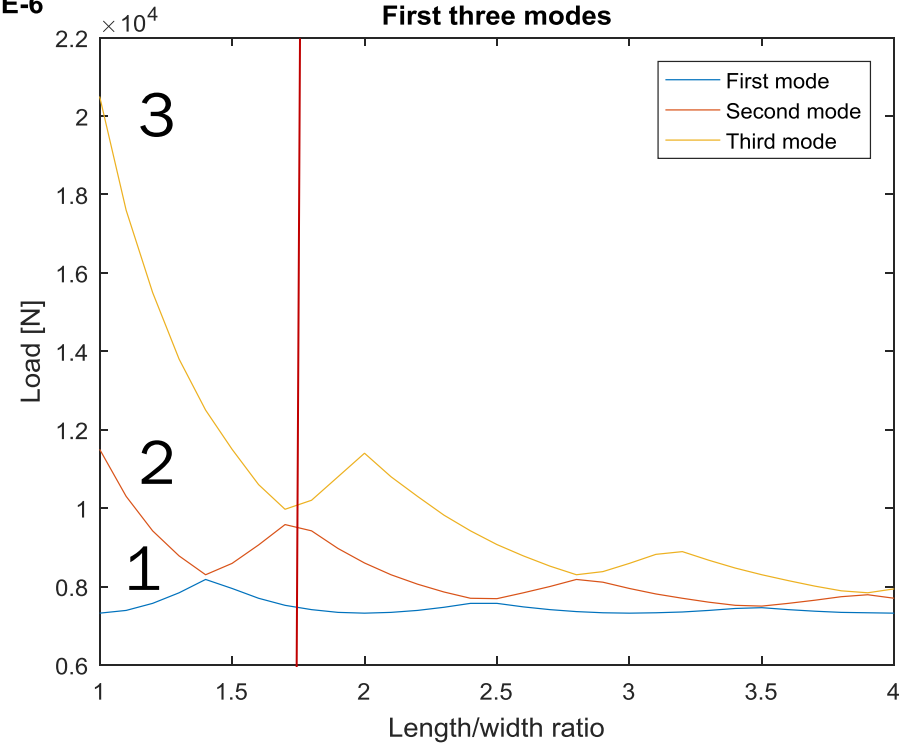


# Switching of modes and related imperfection amplitude

Imperfection amplitude needed to trigger second mode, constant  $\xi_1 = 1.5E-6$



First three modes





- Test cases for nonlinear buckling analysis of plates have been investigated
  - Modal interactions have been investigated for specific imperfection shapes: Influence of imperfection amplitude on mode switching depends strongly on how close the modes are to each other
  - Failure mode of the optimized component might be sensitive to the manufacturing tolerances
- Specific test cases can be used as reference cases in the current extensions of the reduced order approach
  - Analyze manufacturing tolerances through probabilistic inputs
  - Determine sensitivity to imperfections in individual manufacturing processes (parameters)

# Thank you for your attention!



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