

PROGRESSIVE DAMAGE ANALYSIS OF COMPOSITE STRUCTURES VIA ONE-DIMENSIONAL CARRERA UNIFIED FORMULATIONS

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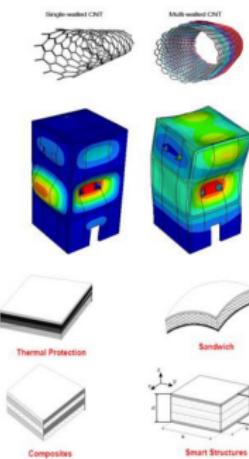
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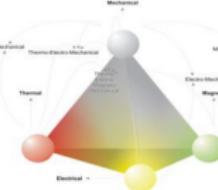
International Conference on Composite Structures, ICCS19
5-9 September 2016, Porto (Portugal)

MUL2 - Our Research Group

MULtilayered structures

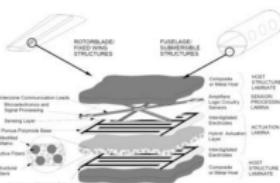
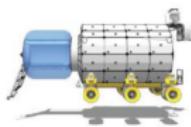
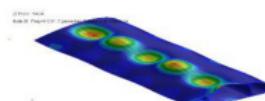


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MUL_{tifield interaction}



Marie Curie Project on Composites



www.fullcomp.net

The FULLCOMP project

- FULLY integrated analysis, design, manufacturing and health-monitoring of COMPosite structures
- The FULLCOMP project is funded by the European Commission under a Marie Skłodowska-Curie Innovative Training Networks grant for European Training Networks (ETN).
- The FULLCOMP partners are:
 - 1 Politecnico di Torino (Italy) - Coordinator
 - 2 University of Bristol (UK)
 - 3 Ecole Nationale Supérieure d'arts et Métiers (Bordeaux, France)
 - 4 Leibniz Universität Hannover (Germany)
 - 5 University of Porto (Portugal)
 - 6 University of Washington (USA)
 - 7 RMIT (Australia)
 - 8 Luxembourg Institute of Technology
 - 9 Elan-Ausy, Hamburg, (Germany)
- FULLCOMP has recruited 12 PhD students who will work in an international framework to develop integrated analysis tools to improve the design of composite structures.
- The full spectrum of the design of composite structures will be dealt with, such as manufacturing, health-monitoring, failure, modeling, multiscale approaches, testing, prognosis, and prognostic. The FULLCOMP research activity is aimed at many engineering fields, e.g. aeronautics, automotive, mechanical, wind energy, and space.
- www.fullcomp.net

Overview

- 1 Description of the Carrera Unified Formulation for refined models (CUF).
- 2 Main 1D CUF capabilities overview (1D Taylor- and Lagrange-based models).
- 3 Numerical examples dealing with different applications (aerospace structures, composites, Bio Structures).
- 4 Introduction to the Component-Wise approach (CW).
- 5 Progressive Damage Model
- 6 Numerical Results

Brief Overview of Beam Refinement Methods and Contributors

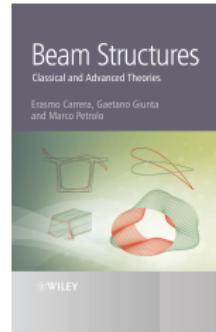
- ① Shear correction factors (Timoshenko, Sokolnikoff, Cowper, Gruttmann, etc.).
- ② Warping functions and Saint-Venant solutions (El Fatmi, Ladéveze, etc.).
- ③ Variational asymptotic method (Berdichevsky, Hodges, Yu, etc.).
- ④ Generalized beam theory (Schardt, Camotim, Silvestre, etc.).
- ⑤ Higher-order models (Washizu, Reddy, Kapania, Carrera, etc.).

1D Advanced Structural Models

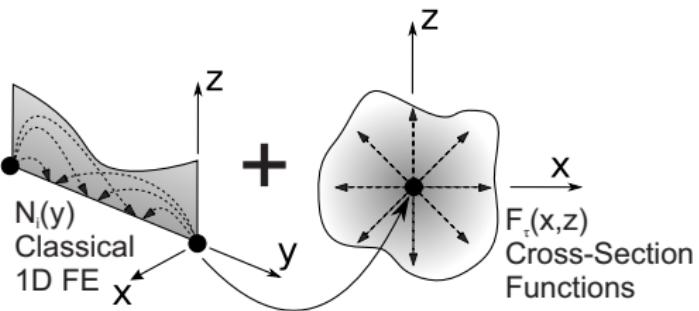
Actual Wing



Our Model



1D Carrera Unified Formulation, CUF - FEM Version



Fundamental Nucleus Equations by CUF

Finite Element Formulation (FEM)

$$\mathbf{u} = F_\tau(x, z) N_i(y) \mathbf{u}_{\tau i} (\textcolor{red}{1D})$$

$$\delta L_{int} = \delta \mathbf{q}_{\tau i}^T \mathbf{K}^{ij\tau s} \mathbf{q}_{sj}$$

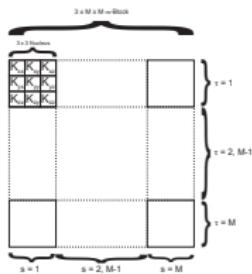
$$\delta L_{ine} = \delta \mathbf{q}_{\tau i}^T \mathbf{M}^{ij\tau s} \ddot{\mathbf{q}}_{sj}$$

$$\delta L_{ext} = \mathbf{P} \delta \mathbf{u}^T$$

i, j : Shape function indexes
(depend on the FE
discretization).

τ, s : Expansion function
indexes (depend on the model
order).

Assembly Technique



Fundamental Nucleus

$$K_{xx}^{ij\tau s} = \tilde{C}_{22} \int_{\Omega} F_{\tau,x} F_{s,x} d\Omega \int_I N_i N_j dy + \dots$$

1D CUF Applications Overview - Books

Thin-Walled and Reinforced Structures

Buckling, Free Vibration and Dynamic Response Analysis

Composite Structures

FGM Structures

Variable Kinematics Models

Axiomatic/Asymptotic Analyses and Best Theory Diagrams

Aeroelasticity

Load Factors and Non-Structural Masses

Rotors and Rotating Blades

Biomechanics

Multifield Analysis

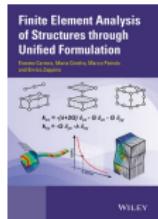
Nanostructures

Analysis of Aerospace Structures via the Component-Wise Approach

Analysis of Civil Structures via the Component-Wise Approach

Component-Wise Approach for the Multiscale Analyses of Composites

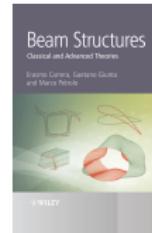
FEM



2D and Smart Structures



1D



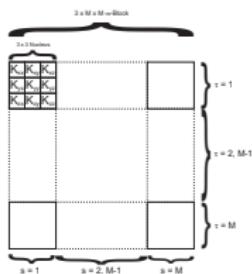
The Taylor CUF 1D Models, TE

$$\begin{aligned}
 u_x &= u_{x_1} + x u_{x_2} + z u_{x_3} + x^2 u_{x_4} + xz u_{x_5} + z^2 u_{x_6} + \dots \\
 u_y &= u_{y_1} + x u_{y_2} + z u_{y_3} + x^2 u_{y_4} + xz u_{y_5} + z^2 u_{y_6} + \dots \\
 u_z &= u_{z_1} + x u_{z_2} + z u_{z_3} + x^2 u_{z_4} + xz u_{z_5} + z^2 u_{z_6} + \dots
 \end{aligned}$$

$\underbrace{N=0}_{\tau=1}$ $\underbrace{N=1}_{\tau=2, \tau=3}$ $\underbrace{N=2}_{\tau=4, \tau=5, \tau=6}$
 3 DOFs 9 DOFs 18 DOFs

Classical models, such as [Timoshenko](#), can be obtained as particular cases of the linear models.

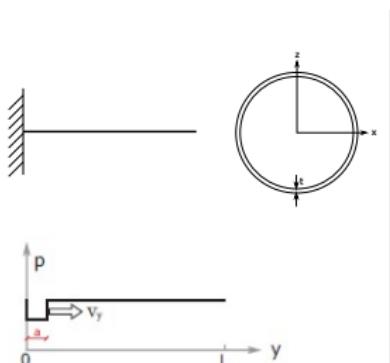
Assembly Technique



Fundamental Nucleus

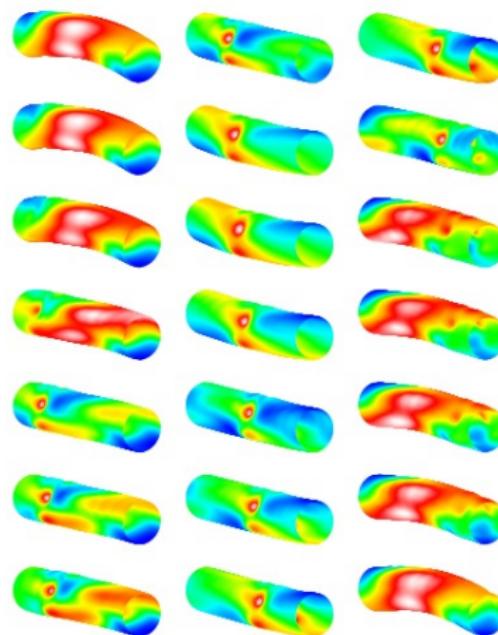
$$K_{xx}^{ijrs} = \tilde{C}_{22} \int_{\Omega} F_{\tau,x} F_{s,x} d\Omega \int_I N_i N_j dy + \dots$$

Wave propagation in a Thin-Walled Cylinder



E. Carrera, A. Varello,
Dynamic Response of
Thin-Walled Structures by
Variable Kinematic
One-Dimensional Models
Journal of Sound and
Vibration, 331(24), pp.
5268-5282, 2012.

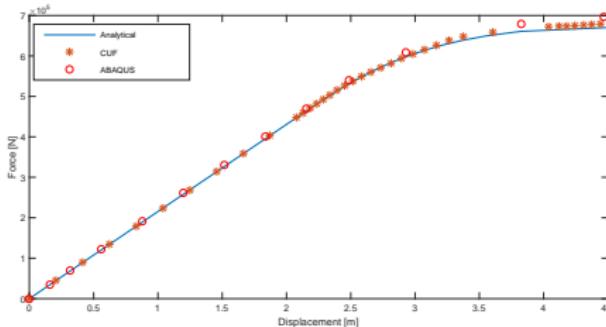
N = 8 beam model, 6000 DOFs



Plastic beam under bending load



- Material model : Isotropic von-Mises plasticity
- Ideal elastic-perfect plastic stress-strain relationship
- Plastic strength factor is estimated
- Emphasizes the importance of refined higher order models



Type	NDOF	U_Z at yield load		U_Z at limit load		Plastic Strength Factor	
		Value [m]	Error [%]	Value [m]	Error [%]	Value	Error [%]
Analytical	-	2.080	-	4.622	-	1.5	-
ABQ - 3D	22,590	2.077	0.14	4.281	7.38	1.55	3.57
Taylor							
EBBM	279	2.083	0.16	3.13	32.39	-	-
TBM	279	2.084	0.18	3.13	32.38	-	-
N = 1	279	2.084	0.18	3.13	32.38	-	-
N = 2	558	2.072	0.40	4.21	8.98	1.59	6.00
N = 3	930	2.072	0.39	4.42	4.80	1.52	1.33

Lagrange-Based 1D Models, LE

$$u_x = L_\tau u_{x_\tau}$$

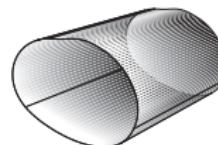
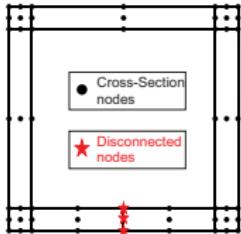
$$u_y = L_\tau u_{y_\tau}$$

$$u_z = L_\tau u_{z_\tau}$$

L9 polynomials - Isoparametric

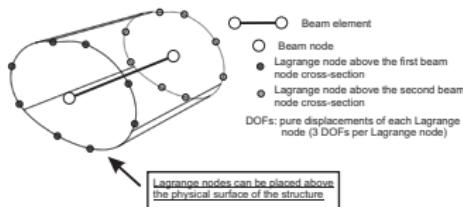
$$L_\tau = \frac{1}{4}(r^2 + r r_\tau)(s^2 + \dots)$$

Cross-Section Elements

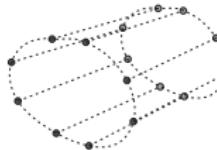


3D Geometry from CAD

LE Modeling

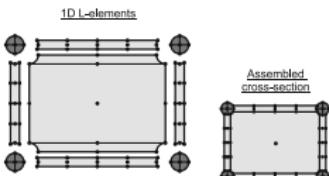
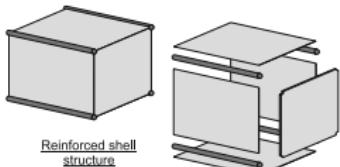


Computational Model

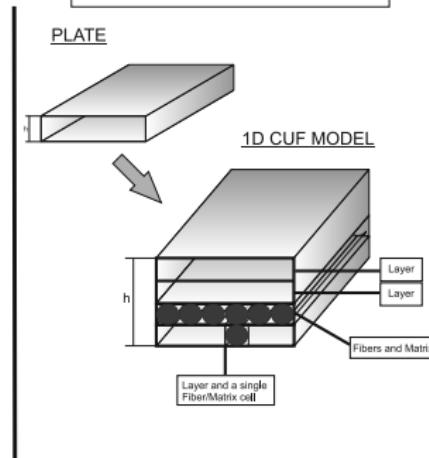


The Component-Wise Approach

Component-Wise approach
for typical aircraft structures

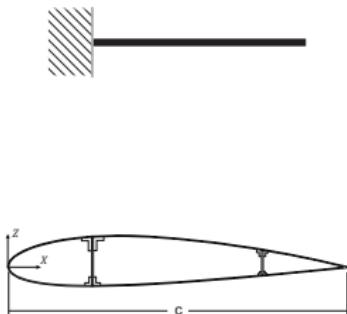


Component-Wise approach
for the multi-scale analysis
of composites



- Only displacements as unknowns.
- Each component of the structure is modeled via beams only.
- No need of reference surfaces. This might be useful in a CAD-FEM interface scenario.
- No need of homogenization techniques.

Aircraft Wing

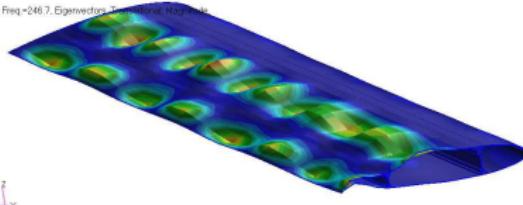


E. Carrera, A. Pagani and M. Petrolo,
Component-wise Method
Applied to Vibration of Wing
Structures,
*Journal of Applied
Mechanics*, 80(4), 2013.
doi:10.1115/1.4007849.

Shell-like modal shapes by means of a BEAM MODEL

Mode 54 - 246.7 Hz

LE Model - NACA
Mode 54 - Freq = 246.7, Eigenvector, Top View, Normal Mode.



Natural frequencies [Hz]

Mode	1D-LE	SOLID
1	4.23	4.22
2	21.76	21.69
3	25.15	24.78
4	31.14	29.18
5	59.26	56.12
6	66.65	62.41
7	74.23	68.77

Computational costs

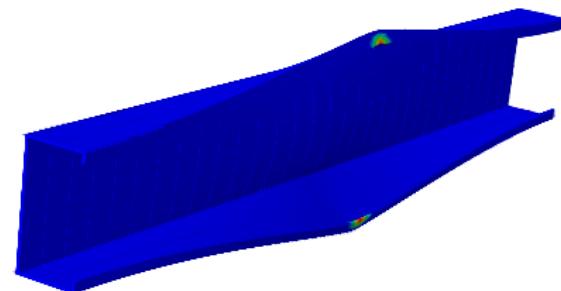
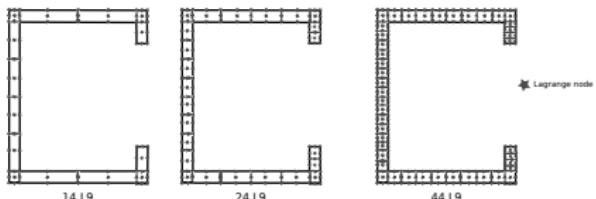
CUF LE \approx **20,000** DOFs

SOLID \approx **190,000** DOFs

Thin-walled lipped channel beam



- Material model : Isotropic von-Mises plasticity
- Ideal elastic-perfect plastic stress-strain relationship



Type	NDOF	Displacement	
		$\lambda = 12.69$	$\lambda = 40.94$
GBT	3,222	13.63	77.6
ABAQUS - Shell	69,948	13.47	74.93
ABAQUS - 3D Brick	1,342,656	13.31	69.92
CUF - CW - (30-B4 elements)			
14L9	23,751	13.17	73.22
24L9	40,131	13.29	75.95
44L9	72,891	13.32	78.34

- 1 Response based on **Continuum Damage Mechanics** and **Hashin's Initiation criterion**
- 2 Three damage variables are introduced : d_f (fiber), d_m (matrix) and d_s (shear) - Tension and Compression handled separately

$$d_I = \frac{\hat{\epsilon}_{I,f} (\hat{\epsilon}_I - \hat{\epsilon}_{I,0})}{\hat{\epsilon}_I (\hat{\epsilon}_{I,f} - \hat{\epsilon}_{I,0})} \quad \forall I \in [ft, fc, mc, mt]$$

- 3 Linear damage evolution law
$$\sigma = \mathbf{C}(d_I) : \epsilon$$
- 4 Mesh objectivity obtained by introducing characteristic length scale

$$\hat{\epsilon}_{I,f} = \frac{2 G_{I,c}}{L_c X_I} \quad \forall I \in [ft, fc, mc, mt]$$

- 5 Viscous regularization scheme introduced to alleviate convergence issues

$$\dot{d}_I^v = \frac{1}{\eta_I} (d_I - d_I^v) \quad \forall I \in [ft, fc, mc, mt]$$

Failure Envelope

Tensile Fiber Mode

$$\hat{\sigma}_{22} = X_T$$

Compressive Fiber Mode

$$|\hat{\sigma}_{22}| = X_C$$

Tensile Matrix Mode ($\sigma_{11} + \hat{\sigma}_{33} > 0$)

$$\frac{1}{Y_T^2} (\hat{\sigma}_{11}^2 + \hat{\sigma}_{33}^2) + \frac{1}{S_T^2} (\hat{\sigma}_{13}^2 - \hat{\sigma}_{11}\hat{\sigma}_{33}) + \frac{1}{S_L^2} (\hat{\sigma}_{12}^2 + \hat{\sigma}_{23}^2) = 1$$

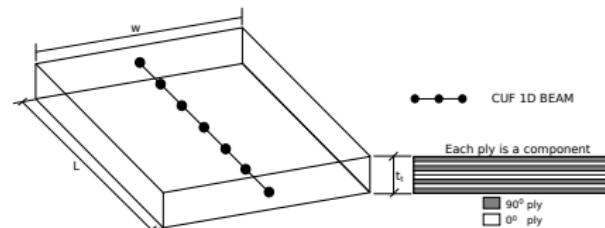
Compressive Matrix Mode ($\hat{\sigma}_{11} + \hat{\sigma}_{33} < 0$)

$$\frac{1}{Y_C} \left[\left(\frac{Y_C}{2S_T} \right)^2 - 1 \right] (\hat{\sigma}_{11} + \hat{\sigma}_{33}) + \frac{1}{4S_T^2} (\hat{\sigma}_{11} + \hat{\sigma}_{33})^2 + \frac{1}{S_T^2} (\hat{\sigma}_{13}^2 + \hat{\sigma}_{11}\hat{\sigma}_{33})^2 + \frac{1}{S_L^2} (\hat{\sigma}_{12}^2 + \hat{\sigma}_{32}^2)^2 = 1$$

Nomenclature : 2 - Fiber direction , 1 & 3 - Transverse direction
 $\hat{\sigma}$ - effective stress

Failure Strength Estimation

- 1 Composite Laminate subjected to uni-axial tension
- 2 Laminate : HTA/6376-C
- 3 Each ply is modeled as a component (5 L9 elements)



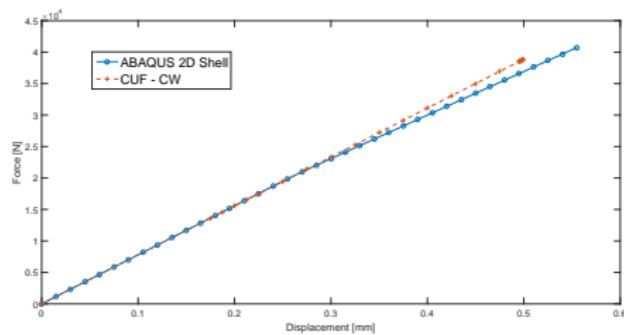
Geometry of the beam

Material properties of HTA /6376-C

E_{11} (GPa)	$E_{22} = E_{33}$ (GPa)	$G_{12} = G_{13}$ (GPa)	G_{23} (GPa)	$\nu_{12} = \nu_{13}$	ν_{23}
139	10	5.2	5.2	0.32	0.51

Orthotropic damage initiation properties for HTA /6376-C

X_T (MPa)	X_C (MPa)	Y_T (MPa)	Y_C (MPa)	S_L (MPa)	S_T (MPa)
2170	1600	73	290	83	83

Stacking Sequence : [90/0]_{2s}

Various stages of damage propagation at fixed end



Fully virgin composite



90° lamina damaged

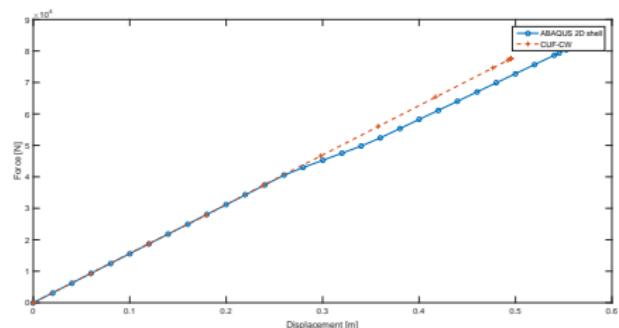


Fully damaged composite

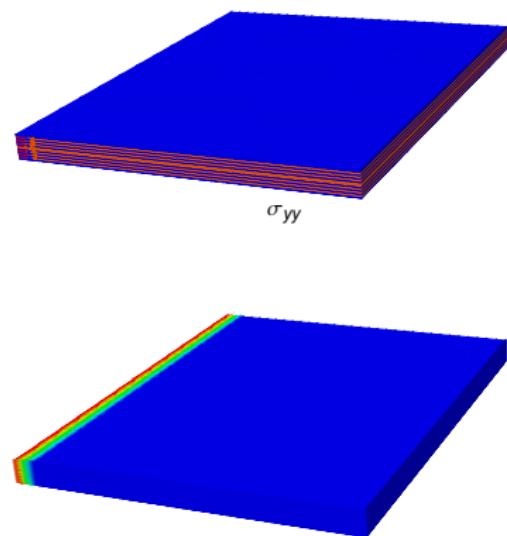
	Experimental	DDM ¹	CUF-CW	ABAQUS-2D shell
Failure Strength	1060	1107.6	1037	1088
DOF	-	900*	17,391	40,000

1. Moure et al., Damage evolution in open-hole laminated composite plates subjected to in-plane loads, 2015

Stacking Sequence : [90/0]_{4s}



	Experimental	DDM ^t	CUF-CW	ABAQUS-2D shell
Failure Strength (MPa)	1110	1118.1	1036.29	1067.73
DOF	-	900*	27,621	40,000



1. Moure et al., Damage evolution in open-hole laminated composite plates subjected to in-plane loads, 2015

Main Conclusions and Perspectives

- Accurate 3D stress fields obtained through CUF is successfully employed for damage propagation analysis
- Framework is able to predict the failure strength of composites (un-notched) within acceptable limits
- Computationally efficient higher-order non-linear finite element framework is developed

Future extensions

- Failure strength estimation of notched composites
- Component wise (fiber and matrix) damage propagation
- Extension of damage analysis for explicit scheme
- Inclusion of other damage initiation criteria

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